## Angular contact ball bearings

Double row





38..-B, 30..-B, 32..-B, 33..-B, 32..-BD, 33..-BD, 32, 33

38..-B, 30..-B, 32..-BD, 33..-BD; with seal 2HRS, 2RS, 2RZ, 2Z

#### d = 90 - 110 mm

Main dimensions		Basic load ratings		Fatigue limit load	Limiting speed	Speed rating	Mass	Designation	
d	D	В	dyn. C <sub>r</sub>	stat. C <sub>0r</sub>	C <sub>ur</sub>	n <sub>G</sub>	n <sub>ðr</sub>	m	<ul> <li>&gt; 326 2.12</li> <li>&gt; 327 2.13</li> <li>X-life &gt; 320</li> </ul>
			Ν	N	N	min <sup>-1</sup>	min <sup>-1</sup>	$\approx$ kg	
90	115	19	27 000	35 500	2 000	5 900	3 5 5 0	0,41	3818-B-TVH
	115	19	27 000	35 500	2 000	2750	-	0,422	3818-B-2RS-TVH
	160	52,4	142 000	142 000	6 500	4 100	4 500	3,8	3218-B-TVH
	160	52,4	142 000	142 000	6 500	2 370	-	4	3218-B-2RS-TVH
	160	52,4	143 000	172 000	8 000	4 350	4 5 5 0	4,14	3218
	190	73	226 000	247 000	11 600	4850	3 4 5 0	10,4	3318-DA-MA
	190	73	260 000	295 000	13 000	3 650	3 350	9,14	3318
95	170	55,6	161 000	193 000	8 800	5 300	4 350	5,06	3219-M
	200	77,8	270 000	315 000	13 500	4 4 5 0	3 200	11,2	3319-M
100	125	19	28 000	39 000	2 1 2 0	5 300	3 1 0 0	0,45	3820-B-TVH
	125	19	28 000	39 000	2120	2 470	-	0,463	3820-B-2RS-TVH
	180	60,3	185 000	173 000	7 600	3 650	4 400	5,4	3220-B-TVH
	180	60,3	185 000	173 000	7 600	2750	4 400	5,5	3220-B-2Z-TVH
	180	60,3	185 000	173 000	7 600	2 2 1 0	-	5,5	3220-B-2RS-TVH
	180	60,3	186 000	235 000	10 300	3 7 5 0	4 0 5 0	5,98	3220
	215	82,6	285 000	340 000	14100	4 200	3 000	14	3320-M
	215	82,6	260 000	305 000	13 600	4 300	3 0 5 0	15,6	3320-DA-MA
105	190	65,1	215 000	270 000	11 400	4 600	3 850	7,4	3221-M
110	200	69,8	236 000	290 000	12 200	4 4 0 0	3 7 5 0	9,03	3222-M
	240	92,1	330 000	425 000	16800	3 7 5 0	2 6 5 0	20	3322-M
	240	92,1	310 000	385 000	17 400	3 800	2 700	21,8	3322-DA-MA

medias https://www.schaeffler.de/std/1F9A



32..-B, 33..-B; with seal 2RSR, 2Z



33..-DA; split inner ring



Mounting dimensions

Dimensions							Nominal contact angle	Mounting dimensions		
d	r	D <sub>1</sub>	D <sub>2</sub>	d <sub>1</sub>	d <sub>2</sub>	a	α	d <sub>a</sub>	D <sub>a</sub>	r <sub>a</sub>
	min.	~	~	~	~	~	0	min.	max.	max.
90	1	106,6	-	98,4	-	55	25	94,6	110,4	1
	1	-	107,2	-	96,2	55	25	94,6	110,4	1
	2	141,6	-	116,4	-	81,4	25	104	146	2
	2	-	145,2	-	112,1	81,4	25	104	146	2
	2	143,7	-	115,6	-	112,5	35	104	146	2
	3	166,2	-	131,9	-	177	45	104	176	2,5
	3	168,2	-	126,1	-	136	35	104	176	2,5
95	2,1	152,8	-	122,2	-	119,8	35	107	158	2,1
	3	177,3	-	133	-	143,3	35	109	186	2,5
100	1	117,9	-	109,5	-	60,2	25	104,6	120,4	1
	1	-	118,5	-	107,3	60,2	25	104,6	120,4	1
	2,1	155,7	-	124,7	-	91,3	25	112	168	2,1
	2,1	-	157,4	-	121,3	91,3	25	112	168	2,1
	2,1	-	157,4	-	121,3	91,3	25	112	168	2,1
	2,1	163,7	-	131	-	127,4	35	112	168	2,1
	3	188,7	-	142,5	-	153,3	35	114	201	2,5
	3	187,1	-	147,5	-	197,5	45	114	201	2,5
105	2,1	172,1	-	138	-	134,7	35	117	178	2,1
110	2,1	180,1	-	143,3	-	143,5	35	122	188	2,1
	3	209,6	-	161,5	-	170,5	35	124	226	2,5
	3	207,3	-	164,5	-	221	45	124	226	2,5



Matrix for bearing preselection \_\_\_\_\_\_353

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## Matrix for bearing preselection

The matrix gives an overview of the types and design features.

It can be used to make a preliminary assessment of whether a bearing is fundamentally suitable for the envisaged application.

The additional information provided in the product chapter (see column "detailed information") and in the Technical principles must, however, be observed in addition to this overview in selection of the bearing.

Design featu	res and suitability		Four point contact bearings		
+++ extrem ++ highly + suitab (+) suitab - not su ✓ availal	ely suitable suitable le le with restrictions itable/not applicable ble		with/without retaining slots	detailed information	
Load	radial		Fr	(+)	> 354 > 356   1.2
carrying capacity	avial one direction				>256 1 2
			-Fa	++	> 550 1.2
	axial, both directions		Fa	++	▶356 1.2
	moments		м	(+)	▶356 1.2
Compen- sation of	static			-	▶356 1.3
angular mis- alignments	dynamic		$\triangleleft$	-	▶356 1.3
Bearing design	cylindrical bore			1	▶354 1.1
	tapered bore			-	
	separable		<b>F</b>	1	▶363 1.17
Lubrication	greased		. 10	_	▶357 1.4
Sealing	open		$\mathbb{M}$	1	▶357 1.5
	non-contact			-	▶357 1.5
	contact			-	▶357 1.5
Operating ter	nperature in °C	from to	<b>I</b>	-30 +150 <sup>1)</sup>	▶358 1.8
Suitability for	high speeds		On	(+)	▶357 1.6
	high running accuracy		11	(+)	<ul> <li>▶ 360 1.11</li> <li>▶ 115</li> </ul>
	low-noise running			(+)	▶357 1.7
	high rigidity		δ	+	▶54
	reduced friction			+	▶56
	length compensation within bearing non-locating bearing arrangement				
					▶141
	locating bearing arrangement	*	++	▶141	
X-life bearing	5		X-life	1	▶355
Bearing bore	d in mm	from to	$\bigcirc$	17 200 <sup>2)</sup>	▶366
Product table	S		<u> </u>		

<sup>1)</sup> Valid for bearings

with brass cages,  $D \leq 240 \text{ mm}$ 

<sup>2)</sup> Larger catalogue bearings GL 1



## Bearing design

Design variants

Four point contact bearings are available as:

- bearings of basic design > 355  $\bigcirc 2$
- bearings with retaining slots in the outer ring >355  $\bigcirc 3$
- X-life bearings ► 355

#### Bearings of basic design

Four point contact bearings are single row, non-self-retaining radial ball bearings. They are similar in their structure to single row radial angular contact ball bearings; the raceways on the inner rings are, however, designed such that they can support axial loads in both directions  $> 355 | \boxdot 2$  and > 356 | 1.2. The centre points of curvature of the arc-shaped raceways on the inner and outer ring are offset relative to each other in such a way that the balls are in contact with the bearing rings at four points under radial load  $> 355 | \boxdot 2$  and > 356 | 1.2.

These bearings have solid outer rings, split inner rings and ball and cage assemblies with brass or polyamide cages > 359 [1.9. The two-piece inner ring allows a large complement of balls to be accommodated in the bearing. The inner ring halves are matched to the particular bearing and must not be interchanged with those of other bearings of the same size. In an axial direction, four point contact bearings are considerably narrower than, for example, double row angular contact ball bearings.

© Comparable, in terms of product design, with single row radial angular contact ball bearings

Smaller axial section height than double row angular contact ball bearings

#### Four point contact bearing of basic design

#### $\alpha = nominal \ contact \ angle$

- M<sub>1</sub>, M<sub>2</sub> = centres of curvature of outer ring raceway
- $F_r = radial \ load$
- $F_a = axial \ load$
- Four point contact bearing, split inner ring, without retaining slots in the outer ring
- (2) Raceway geometry



#### Bearings with retaining slots in the outer ring

Four point contact bearings are often combined with a radial bearing and used as an axial bearing with radial clearance in a housing >355  $\bigcirc$  3, >362 | 1.16. For quick and secure location of the bearings in the housing, larger bearings therefore have two retaining slots in one end face of the outer ring offset by 180° >355  $\bigcirc$  3. Locking pins engage in these retaining slots and locate the outer ring in the housing.



Solution States Sta

in the housing

- Cylindrical roller bearing (radial bearing)
- (2) Four point contact bearing with retaining slots in outer ring (axial bearing, outer ring not radially retained)
- ③ Locking pin for location of outer ring



#### X-life premium quality

Four point contact bearings are available in certain sizes as X-life bearings. These bearings exhibit considerably higher performance than standard four point contact bearings > 356  $\bigcirc$  4. This is achieved, for example, through the modified internal construction, higher surface quality of the contact surfaces and optimised cage design, as well as through the improved quality of the steel and rolling elements.

#### Advantages

The technical enhancements offer a range of advantages, such as:

- a more favourable load distribution in the bearing and thus a higher dynamic load carrying capacity of the bearings > 290 \@ 6
- quieter running
- running with reduced friction and greater energy efficiency
- Iower heat generation in the bearing
- higher possible speeds
- lower lubricant consumption and, consequently, longer maintenance intervals
- a measurably longer operating life
- high operational security
- compact, environmentally-friendly bearing arrangements

Solution States Sta

X-life

## Solution States Sta

Comparison of basic dynamic load rating  $C_r$  – bearing series QJ3..-XL,

bore code 5 to 14, with a bearing

 $C_r$  = basic dynamic load rating

(1) Bore code

which is not of X-life quality

#### 🕾 Suffix XL

⊕\_4

In conclusion, these advantages improve the overall cost-efficiency of the bearing position significantly and thus bring about a sustainable increase in the efficiency of the machine and equipment.

X-life four point contact bearings include the suffix XL in the designation  $> 360 | \boxplus 4$ ,  $> 360 | \oplus 6$  and  $> 366 | \boxplus 1$ .



#### Areas of application

Solution State State

Due to their special technical features, X-life four point contact bearings are highly suitable for bearing arrangements in:

- compressors
- fluid and hydraulic pumps
- automotive chassis and gearboxes
- gearboxes for industrial, rail and wind turbine applications
- agricultural vehicles and equipment

X-life indicates a high product performance density and thus a particularly significant benefit to the customer. Further information on X-life > 10.

### Load carrying capacity

Scapable of supporting high axial loads in both directions Due to the design of the raceways with their high shoulders, the large nominal contact angle of  $\alpha_0 = 35^\circ$  and the large number of rolling elements, four point contact bearings have a very high axial load carrying capacity. They are suitable for alternating, purely axial loads or predominantly axial load. The balls are in contact with the inner ring and outer ring each at one point only, as is the case with a single row angular contact ball bearing under axial load > 355  $\bigcirc 2$ .

The radial load carrying capacity of the bearings is low. If predominantly radial load is present, four point contact bearings should not be used due to the higher friction in the four point contact.

## Compensation of angular misalignments

Sour point contact bearings cannot compensate misalignments



Four point contact bearings are not suitable for the compensation of angular misalignments due to housing deformations or shaft deflections. The possible skewing of the inner ring in relation to the outer ring depends, for example, on the bearing load, the operating clearance and the bearing size, and is very small.

Skewing of the bearing rings increases the running noise, places increased strain on the cages and has a harmful influence on the operating life of the bearings.

© Compatibility with plastic cages

> Observe oil change intervals

4 Lubrication

◎ *Oil or grease lubrication* The bearings are not greased. They must be lubricated with oil or grease. When using bearings with plastic cages, compatibility between the lubricant and the cage material must be ensured if synthetic oils, lubricating greases with a synthetic oil base or lubricants containing a high proportion of EP additives are used.

> Aged oil and additives in the oil can impair the operating life of plastics at high temperatures. As a result, stipulated oil change intervals must be strictly observed.

## Sealing

of an open design

The bearings are Four point contact bearings are supplied without seals. As a result, sealing of the bearing position must be carried out in the adjacent construction. The sealing system should reliably prevent:

- moisture and contaminants from entering the bearing
- the egress of lubricant from the bearing

## Speeds

Higher speeds are Due to the four point contact and resulting higher level of friction. only possible under purely axial load

the speed suitability of the bearings is heavily restricted under radial load. Higher speeds can only be achieved if four point contact ball bearings are subjected to purely axial load.

- Speeds The product tables generally give two speeds for the bearings:
- *in the product tables* **•** the kinematic limiting speed n<sub>G</sub>
  - the thermal speed rating n<sub>θr</sub>

#### Limiting speeds

The limiting speed n<sub>G</sub> is the kinematically permissible speed of the bearing. Even under favourable mounting and operating conditions, this value should not be exceeded without prior consultation with Schaeffler ▶64.

The values in the product tables are valid for oil lubrication. Bearings with solid brass cages have a significantly higher limiting speed for oil lubrication.

<sup>∞</sup> Values for grease For grease lubrication, 75% of the value stated in the product tables lubrication is permissible in each case.

For the grease lubrication of bearings with solid brass cages, 58% of the value stated in the product tables is permissible in each case.

#### **Reference speeds**

 $\otimes n_{\mathcal{H}}$  is used to calculate  $n_{\mathcal{H}}$ 

The thermal speed rating  $n_{\vartheta r}$  is not an application-oriented speed limit, but is a calculated ancillary value for determining the thermally safe operating speed  $n_{\vartheta} \ge 64$ .



The Schaeffler Noise Index (SGI) has been developed as a new feature for comparing the noise level of different bearing types and series. As a result, a noise evaluation of rolling bearings can now be carried out for the first time.

#### **Schaeffler Noise Index**

The SGI value is based on the maximum permissible noise level of a bearing in accordance with internal standards, which is calculated on the basis of ISO 15242. In order that different bearing types and series can be compared, the SGI value is plotted against the basic static load rating C<sub>0</sub>.

This permits direct comparisons between bearings with the same load carrying capacity. The upper limit value is given in each of the diagrams. This means that the average noise level of the bearings is lower than illustrated in the diagram.

The Schaeffler Noise Index is an additional performance characteristic in the selection of bearings for noise-sensitive applications. The specific suitability of a bearing for an application in terms of installation space, load carrying capacity or speed limit for example, must be checked independently of this.

The Noise Index is currently available for the main series. Additional series will be updated and introduced in subsequent publications.

#### Further information:

#### **medias** https://medias.schaeffler.com



0,3 0,15 QJ2..-XL QJ3..-XL SGI 0,03 10 000 40 000 100,000 300,000 Ν 000A351 C<sub>0</sub> -

for four point contact bearings

SGI = Schaeffler Noise Index  $C_0 = basic static load rating$ 

### Temperature range

Substant States Sta

1.8

- The operating temperature of the bearings is limited by:
- the dimensional stability of the bearing rings and rolling elements
- the cage
- the lubricant

Possible operating temperatures of four point contact bearings ▶358 🖽 1.

1 Permissible temperature ranges

Operating	Four point contact bearings				
temperature	with brass cage	with polyamide cage PA66			
J	-30 °C to +150 °C, for D > 240 mm up to +200 °C	-30 °C to +120 °C			



In the event of anticipated temperatures which lie outside the stated values, please contact Schaeffler.



#### 9 Cages

Solid cages made from *Solid* brass and polyamide PA66 are used as standard

Standard cages and additional designs for four point contact bearings > 359  $\blacksquare$  2. Other cage designs are available by agreement. With such cages, however, suitability for high speeds and temperatures as well as the basic load ratings may differ from the values for the bearings with standard cages.

For high continuous temperatures and applications with difficult operating conditions, bearings with brass or sheet steel cages should be used. If there is any uncertainty regarding cage suitability, please consult Schaeffler.



			E	<b></b>
Cage,	cage	suffix,	bore	code

Bearing series	Solid brass cage		Solid cage made from polyamide PA66 TVP		
	standard	also available for	standard	also available for	
	Bore code				
QJ10	12, 17, 19, 21, 22, 24, 26, 30 to 40	-	_	_	
QJ2	up to 08, 10, 13, 16, 17, from 19	09, 11, 12, 14, 15, 18	09, 11,12, 14, 15, 18	08	
QJ3	03, 04, from 10	05 to 09	05 to 09	-	

## 1.10 Internal clearance

#### Axial internal clearance

The standard is CN Four point contact bearings are manufactured as standard with axial internal clearance CN (normal) > 359 3. CN is not stated in the designation.



Certain sizes are also available by agreement with the smaller internal clearance C2 and with the larger internal clearance C3 and C4.

The values for axial internal clearance correspond to DIN 628-4:2008 DIN (ISO 5753-2:2010) ► 359 3. They are valid for bearings which are free from load and measurement forces (without elastic deformation).

3
Axial internal clearance
of four point contact bearings

Nominal bore diameter		Axial internal clearance								
d mm		C2 (Group µm	2) CN (Group N) µm		C3 (Group 3) μm		C4 (Group 4) μm			
over	incl.	min.	max.	min.	max.	min.	max.	min.	max.	
10	18	15	65	50	95	85	130	120	165	
18	40	25	75	65	110	100	150	135	185	
40	60	35	85	75	125	110	165	150	200	
60	80	45	100	85	140	125	175	165	215	
80	100	55	110	95	150	135	190	180	235	
100	140	70	130	115	175	160	220	205	265	
140	180	90	155	135	200	185	250	235	300	
180	220	105	175	155	225	210	280	260	330	

## 🛽 Dimensions, tolerances

#### **Dimension standards**

The main dimensions of four point contact bearings correspond to DIN 628-4:2008. Nominal dimensions of four point contact bearings > 366

#### **Chamfer dimensions**

The limiting dimensions for chamfer dimensions correspond to DIN 620-6:2004. Overview and limiting values > 137 7.11. Nominal value of chamfer dimension > 366

#### **Tolerances**

The tolerances for the dimensional and running accuracy of four point contact bearings correspond to tolerance class Normal in accordance with ISO 492:2014. Tolerance values in accordance with ISO 492 > 124  $\boxplus 8$ .

#### **Retaining slots**

The dimensions and tolerances of the retaining slots are based on ISO 20515:2021 bzw. DIN 628-4:2008.



Suffixes and

corresponding descriptions

### Suffixes

Suffixes describe the design and features of a bearing in more detail.

Suffix	Description of suffix						
C2	Axial internal clearance C2 (smaller than normal)	Special design, available by agreement					
C3	Axial internal clearance C3 (larger than normal)						
C4	Axial internal clearance C4 (larger than C3)						
MPA	Solid brass cage, guided on outer ring	Standard,					
TVP	Solid cage made from glass fibre reinforced polyamide PA66	cage material dependent on bearing series and bore code					
XL	X-life bearing	Standard, dependent on bore code and bearing type					
N2	Two retaining slots in outer ring	Standard for larger bearings					

## **1.13** Structure of bearing designation

With *medias* interchange, equivalent Schaeffler bearing designations can be determined for bearing designations from other rolling bearing manufacturers *https://www.schaeffler.de/std/1B69*.

The designation of bearings follows a set model. For an example
360 ♀ 6. The composition of designations is subject to DIN 623-1
102 ♀ 10.



Example of composition of bearing designation

> Four point contact bearing with two retaining slots in the outer ring: designation structure

## 1.14 Dimensioning

#### Equivalent dynamic bearing load

 $\otimes P = F_r$  under purely radial load of constant magnitude and direction

Sector Secto for combined load and various load cases

dynamic bearing load P.  $\mathbb{S}F_{o}/F_{r} \leq 0.95$  or The calculation of P is dependent on the load ratio  $F_{a}/F_{r}$  and the  $F_0/F_r > 0.95$  factor 0.95 > 361 f 1 and > 361 f 2.

The basic rating life equation  $L = (C_r/P)^p$  used in the dimensioning of

mined for the rating life calculation that (in relation to the rating life) represents an equivalent load. This force is known as the equivalent

bearings under dynamic load assumes a load of constant magnitude and direction. In radial bearings, this is a purely radial load F<sub>r</sub>. If this condition

is met, the bearing load  $F_r$  is used in the rating life equation for P (P =  $F_r$ ). If this condition is not met, a constant radial force must first be deter-

Equivalent dynamic load

$$\frac{F_a}{F_r} \leq 0.95 \implies P = F_r + 0.66 \cdot F_a$$

\_f]**2** Equivalent dynamic load

$\frac{F_a}{F_r} > 0.9$	95 ⇒	$P = 0,6 \cdot F_r + 1,07 \cdot F_a$
P	N	Equivalent dynamic bearing load
F <sub>r</sub>	N	Radial load

Axial load.

check the static load safety factor  $S_0 > 361 \pm 4$ .

#### Equivalent static bearing load

For four point contact bearings under static load  $> 361 \pm 13$ .

\_f]**3** Equivalent static load Legend

$P_0 = F_{0r} + 0.58 \cdot F_{0a}$							
P <sub>0</sub>	Ν	Equivalent static bearing load					
$F_{0r}, F_{0a}$	Ν	Largest radial or axial load present (maximum load).					

#### Static load safety factor

 $S_0 = C_0/P_0$  In addition to the basic rating life L (L<sub>10h</sub>), it is also always necessary to

\_t\_14 Static load safety factor

_ƒ <b>∐ 4</b> ety factor	$S_0 = \frac{C_0}{P_0}$	<u>0</u> D	
Legend	S <sub>0</sub>	–	Static load safety factor
	C <sub>0</sub>	N	Basic static load rating
	P <sub>0</sub>	N	Equivalent static bearing load.

## Minimum load

damage due to slippage, a minimum axial load

In order to prevent In order to ensure low friction in the bearing, especially at high speeds, a minimum axial load is required. In order to prevent an excessive increase in friction in the bearing, the axial force should be sufficiently of  $F_a \ge 1, 2 \cdot F_r$  is required high that the rolling elements are in contact with the inner and outer ring raceway at only one point. This is ensured if  $F_a \ge 1, 2 \cdot F_r$ .

\_f]**1** 

Legend

Fa

N

## **1.16** Design of bearing arrangements

Support bearing rings over their entire circumference and width

Subset as axial bearing If a four point contact bearing is used as a pure axial bearing, the outer ring must have a large radial clearance in the housing, in order that the bearing is not subjected to radial load >355  $\bigcirc$  3.

> In order to allow full utilisation of the load carrying capacity of the bearings and thus also achieve the requisite rating life, the bearing rings must be rigidly and uniformly supported by means of contact surfaces over their entire circumference and over the entire width of the raceway (not applicable to bearings with radially relieved outer rings). The seating and contact surfaces should not be interrupted by grooves, holes or other recesses. The accuracy of mating parts must meet specific requirements ▶ 363 5 to ▶ 363 7.

#### Radial location of bearings – fit recommendations

Solution Secure radial location. ⊗ tiaht fits are necessary

In addition to supporting the rings adequately, the bearings must also be securely located in a radial direction, to prevent creep of the bearing rings on the mating parts under load. This is generally achieved by means of tight fits between the bearing rings and the mating parts. If the rings are not secured adequately or correctly, this can cause severe damage to the bearings and adjacent machine parts. Influencing factors, such as the conditions of rotation, magnitude of the load, internal clearance, temperature conditions, design of the mating parts and the mounting and dismounting options must be taken into consideration in the selection of fits.

If shock type loads occur, tight fits (transition fit or interference fit) are required to prevent the rings from coming loose at any point. Clearance, transition or interference fits  $> 152 \equiv 6$  and  $> 160 \equiv 7$ .

The following information provided in Technical principles must be taken into consideration in the design of bearing arrangements:

- **c**riteria for selection of fits > 146
- conditions of rotation > 147
- tolerance classes for cylindrical shaft seats (radial bearings) ▶149 3
- **shaft fits**  $> 152 \square 6$
- tolerance classes for bearing seats in housings (radial bearings) ▶150 🖽 4
- housing fits  $> 160 \square 7$

For location of the bearings in the housing by means of retaining slots and

#### Axial location of bearings - location methods

As a tight fit alone is not normally sufficient to also locate the bearing rings securely on the shaft and in the housing bore in an axial direction, this must usually be achieved by means of an additional axial location or retention method. The axial location of the bearing rings must be matched to the type of bearing arrangement. Shaft and housing shoulders, housing covers, nuts, spacer rings and retaining rings etc., are fundamentally suitable ► 355 🖓 3.

#### Dimensional, geometrical and running accuracy of the bearing seats

The accuracy of the bearing seat on the shaft and in the housing should correspond to the accuracy of the bearing used. For four point contact bearings with the tolerance class Normal, the shaft seat should correspond to a minimum of standard tolerance grade IT6 and the housing seat to a minimum of IT7. Guide values for the geometrical and positional tolerances of bearing seating surfaces  $> 363 | \pm 5$ . Tolerances  $t_1$  to  $t_3$  in accordance with > 170  $\bigcirc 11$ . Numerical values for IT grades  $> \overline{3}63 \equiv 6$ .

Substant States Sta by means of retaining slots locking pin >355  $\bigcirc$  3.

Some services to be ™
Some services of the service of the services securely located in an axial direction

> *∞A minimum of IT6* should be provided for the shaft seat and a minimum of IT7 for the housing seat





Guide values for the geometrical and positional tolerances of bearing seating surfaces

5 🖽

Bearing tolerance class		Bearing seating	Standard tolerance grades to ISO 286-1 (IT grades)						
to ISO 492	to DIN 620	surface	Diameter tolerance	Roundness tolerance	Parallelism tolerance	Total axial runout tolerance of abutment shoulder			
				t <sub>1</sub>	t <sub>2</sub>	t <sub>3</sub>			
Normal	PN (P0)	Shaft Housing	IT6 (IT5)	Circumfer- ential load IT4/2	Circumfer- ential load IT4/2	IT4			
				Point load IT5/2	Point load IT5/2				
			IT7 (IT6)	Circumfer- ential load IT5/2	Circumfer- ential load IT5/2	IT5			
				Point load IT6/2	Point load IT6/2				

Numerical values for ISO standard tolerances (IT grades) to ISO 286-1:2010

	Nominal dimension in mm										
ade	over	10	18	30	50	80	120	180	250		
IT gr	incl.	18	30	50	80	120	180	250	315		
	Values in $\mu$ m										
IT4		5	6	7	8	10	12	14	16		
IT5		8	9	11	13	15	18	20	23		
IT6		11	13	16	19	22	25	29	32		
IT7		18	21	25	30	35	40	46	52		

#### Roughness of cylindrical bearing seating surfaces

Sa must not be too high

The roughness of the bearing seats must be matched to the tolerance class of the bearings. The mean roughness value Ra must not be too high, in order to maintain the interference loss within limits. The shafts must be ground, while the bores must be precision turned. Guide values as a function of the IT grade of bearing seating surfaces  $> 363 | \blacksquare 7$ .

	1001
Roughness values	oft
for cylindrical bearing seating	d ([
surfaces – guide values	mm

**7** 

Nominal diameter of the bearing seat d (D)		Recommended mean roughness value for ground bearing seats Ramax						
mm		μm						
		Diameter tolerance (IT grade)						
over	incl.	IT7	IT6	IT5	IT4			
-	80	1,6	0,8	0,4	0,2			
80	500	1,6	1,6	0,8	0,4			

The contact surfaces for the rings must be of sufficient height

#### Mounting dimensions for the contact surfaces of bearing rings

The mounting dimensions of the shaft and housing shoulders, and spacer rings etc., must ensure that the contact surfaces for the bearing rings are of sufficient height. However, they must also reliably prevent rotating parts of the bearing from grazing stationary parts. Proven mounting dimensions for the radii and diameters of the abutment shoulders > 366 . These dimensions are limiting dimensions (maximum or minimum dimensions); the actual values should not be higher or lower than specified.

## Mounting and dismounting

The mounting and dismounting options for four point contact bearings, by thermal, hydraulic or mechanical methods, must be taken into consideration in the design of the bearing position. self-retaining, they are easy to mount

Solling bearings must be handled with great care

S As the bearings are not Four point contact bearings are not self-retaining. As a result, the outer ring with the ball and cage assembly can be mounted separately from the two inner ring halves > 354 1.1. This gives simplified mounting of the bearings.

#### Schaeffler Mounting Handbook

Rolling bearings are well-proven precision machine elements for the design of economical and reliable bearing arrangements, which offer high operational security. In order that these products can function correctly and achieve the envisaged operating life without detrimental effect, they must be handled with care.



The Schaeffler Mounting Handbook MH 1 gives comprehensive information about the correct storage, mounting, dismounting and maintenance of rotary rolling bearings https://www.schaeffler.de/std/1D53. It also provides information which should be observed by the designer, in relation to the mounting, dismounting and maintenance of bearings, in the original design of the bearing position. This book is available from Schaeffler on request.

## 1.18 Legal notice regarding data freshness

Some states the second sec of products may also result in technical changes to catalogue products

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#### Link to electronic product catalogue

The following link will take you to the Schaeffler electronic product catalogue: https://medias.schaeffler.com.

## 1.19 Further information

In addition to the data in this chapter, the following chapters in Technical principles must also be observed in the design of bearing arrangements:

- Determining the bearing size > 34
- Rigidity > 54
- Friction and increases in temperature ► 56
- Speeds ≥64
- Bearing data ►97
- Lubrication > 70
- Sealing ≥ 185
- Design of bearing arrangements > 141
- Mounting and dismounting > 194

 $\bigotimes$ 





N2 variant

#### d = 17 – 85 mm

Main dimensions		Basic load ratings		Fatigue limit load	Limiting speed	Speed rating	Mass	Designation	
d	D	В	dyn. C <sub>r</sub>	stat. C <sub>0r</sub>	C <sub>ur</sub>	n <sub>G</sub>	n <sub>ðr</sub>	m	<ul> <li>▶ 360  1.12</li> <li>▶ 360  1.13</li> <li>X-life &gt; 355</li> </ul>
			Ν	Ν	Ν	min <sup>-1</sup>	min <sup>-1</sup>	$\approx$ kg	
17	47	14	24 500	15 100	1130	29 500	12000	0,148	QJ303-XL-MPA
20	52	15	31 000	19600	1 360	26 000	10700	0,184	QJ304-XL-MPA
25	52	15	26 000	18 800	1 300	25 500	12 300	0,171	QJ205-XL-MPA
	62	17	46 500	31 500	2180	14100	8 800	0,256	QJ305-XL-TVP
30	62	16	37 500	27 500	1 930	21 100	10 200	0,254	QJ206-XL-MPA
	72	19	61 000	43 000	3 000	11 900	7 600	0,379	QJ306-XL-TVP
35	72	17	45 000	35 500	2 470	18000	8 500	0,359	QJ207-XL-MPA
	80	21	65 000	51 000	3 500	10800	7 000	0,516	QJ307-XL-TVP
40	80	18	58 000	46 500	3 2 5 0	10 600	7 500	0,399	QJ208-XL-TVP
	90	23	90 000	69 000	4750	9 300	6 200	0,695	QJ308-XL-TVP
45	85	19	66 000	57 000	4 0 0 0	9800	6 900	0,467	QJ209-XL-TVP
	100	25	107 000	83 000	6 200	8 300	5 700	0,934	QJ309-XL-TVP
50	90	20	62 000	56 000	3 950	13900	6700	0,609	QJ210-XL-MPA
	110	27	115 000	92 000	6800	11 300	5 400	1,39	QJ310-XL-MPA
55	100	21	81 000	76 000	5 300	8 200	5 800	0,697	QJ211-XL-TVP
	120	29	133 000	108 000	8100	10 300	5 000	1,76	QJ311-XL-MPA
60	95	18	51 000	52 000	3 600	13100	5 800	0,42	QJ1012-XL-MPA
	110	22	98 000	93 000	6 6 0 0	7 400	5 300	0,889	QJ212-XL-TVP
	130	31	152 000	126 000	9 200	9 500	4700	2,2	QJ312-XL-MPA
65	120	23	106 000	104 000	7 200	10 300	4 900	1,27	QJ213-XL-MPA
	140	33	171 000	145 000	10800	8 7 0 0	4 4 5 0	2,71	QJ313-XL-MPA
70	125	24	123 000	122 000	9 300	6 500	4 600	1,19	QJ214-XL-TVP
	150	35	198 000	165 000	11800	8100	4 200	3,29	QJ314-XL-MPA
75	130	25	129 000	130 000	9 300	6 200	4 4 5 0	1,34	QJ215-XL-TVP
	160	37	229 000	204 000	14400	7 600	3 900	3,95	QJ315-XL-N2-MPA
80	140	26	136 000	137 000	9700	8 600	4 2 5 0	1,84	QJ216-XL-MPA
	170	39	226 000	220 000	11100	7 000	3750	4,65	QJ316-N2-MPA
85	130	22	86 000	95 000	6 400	9 200	4 2 5 0	1,11	QJ1017-XL-N2-MPA
	150	28	158 000	160 000	11100	8 0 0 0	4 0 5 0	2,3	QJ217-XL-MPA
	180	41	248 000	255 000	12800	6 6 0 0	3 5 5 0	5,53	QJ317-N2-MPA

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d = 17 – 85 mm



Mounting dimensions

Dimensions									Mounting dimensions		
d	r	D <sub>1</sub>	d <sub>1</sub>	a	a <sub>n</sub>	b <sub>n</sub>	r <sub>n</sub>	d <sub>a</sub>	D <sub>a</sub>	r <sub>a</sub>	
	min.	~	~	~				min.	max.	max.	
17	1	36,4	27,8	22	-	-	-	22,6	41,4	1	
20	1,1	41,4	30,6	26	-	-	-	27	45	1	
25	1	43,1	33,9	27	-	-	-	31	46	1	
	1,1	49,5	37,5	31	-	-	-	32	55	1	
30	1	50,7	40,4	32	-	-	-	36	56	1	
	1,1	58	44	36	-	-	-	37	65	1	
35	1,1	59,1	48	38	-	-	-	42	65	1	
	1,5	64,8	50,8	41	-	-	-	44	71	1,5	
40	1,1	66,8	53,7	42	-	-	-	47	73	1	
	1,5	73,4	56,7	46	-	-	-	49	81	1,5	
45	1,1	72	58,5	45	-	-	-	52	78	1	
	1,5	81,7	63,4	51	-	-	-	54	91	1,5	
50	1,1	76,4	63,7	49	-	-	-	57	83	1	
	2	89,6	70,5	56	-	-	-	61	99	2	
55	1,5	84,7	70,4	54	-	-	-	64	91	1,5	
	2	97,8	77,2	61	-	-	-	66	109	2	
60	1,1	83,1	72,4	54	-	-	-	66	89	1	
	1,5	93	77,1	60	-	-	-	69	101	1,5	
	2,1	106,9	84,2	67	-	-	-	72	118	2,1	
65	1,5	101,5	84,2	65	-	-	-	74	111	1,5	
	2,1	114,4	91	72	-	-	-	77	128	2,1	
70	1,5	106,3	89,1	68	-	-	-	79	116	1,5	
	2,1	123,6	97,7	77	-	-	-	82	138	2,1	
75	1,5	111,5	93,9	72	-	-	-	84	121	1,5	
	2,1	131	104,4	82	10,1	8,5	2	87	148	2,1	
80	2	119,6	100,9	77	-	-	-	91	129	2	
	2,1	140,8	110,7	88	10,1	8,5	2	92	158	2,1	
85	1,1	114,8	101,1	75	5	6,5	0,5	91	124	1	
	2	128,6	107,6	82	-	-	-	96	139	2	
	3	148,7	117,9	93	11,7	10,5	2	99	166	2,5	





N2 variant

Mounting dimensions

#### d = 90 - 200 mm

Main dimensions		Basic load ratings		Fatigue limit load	Limiting speed	Speed rating	Mass	Designation	
d	D	В	dyn. C <sub>r</sub>	stat. C <sub>0r</sub>	C <sub>ur</sub>	n <sub>G</sub>	n <sub>ðr</sub>	m	> 360   1.12 > 360   1.13 X-life > 355
			Ν	N	N	min <sup>-1</sup>	min <sup>-1</sup>	$\approx$ kg	
90	160	30	189 000	198 000	12900	4950	3 7 5 0	2,35	QJ218-XL-N2-TVP
	190	43	265 000	285 000	13 300	6 3 0 0	3 3 5 0	6,31	QJ318-N2-MPA
95	145	24	106 000	121 000	7 600	8 200	3 850	1,56	QJ1019-XL-N2-MPA
	170	32	190 000	212 000	10 400	7 0 0 0	3 700	3,41	QJ219-N2-MPA
	200	45	285 000	315 000	14 500	5 900	3 2 5 0	7,45	QJ319-N2-MPA
100	180	34	224 000	241 000	11 500	6 6 0 0	3 5 5 0	4,02	QJ220-N2-MPA
	215	47	325 000	365 000	16800	5 400	3 000	9,04	QJ320-N2-MPA
105	160	26	126 000	145 000	8 700	7 400	3 5 5 0	2,04	QJ1021-XL-N2-MPA
	190	36	233 000	255 000	12 000	6 200	3 4 5 0	4,81	QJ221-N2-MPA
110	170	28	138 000	184 000	8 100	6900	3 350	2,52	QJ1022-N2-MPA
	200	38	249 000	285 000	12700	5 900	3 350	5,66	QJ222-N2-MPA
	240	50	345 000	415 000	18 000	4950	2 700	12,2	QJ322-N2-MPA
120	180	28	145 000	200 000	8 500	6 500	3 100	2,71	QJ1024-N2-MPA
	215	40	285 000	340 000	15 100	5 400	3 0 5 0	6,74	QJ224-N2-MPA
	260	55	385 000	485 000	19900	4 5 5 0	2 480	15,6	QJ324-N2-MPA
130	230	40	295 000	370 000	15800	5 100	2 800	7,66	QJ226-N2-MPA
	280	58	425 000	570 000	22 200	4 200	2 2 2 0	19,2	QJ326-N2-MPA
140	250	42	315 000	420 000	16900	4700	2 600	9,69	QJ228-N2-MPA
	300	62	470 000	660 000	25 500	3 900	2 0 3 0	23,2	QJ328-N2-MPA
150	225	35	205 000	295 000	11 200	5 100	2 650	6,17	QJ1030-N2-MPA
	270	45	350 000	485 000	18 900	4 3 5 0	2 360	12,2	QJ230-N2-MPA
	320	65	510 000	730 000	26 000	3 6 5 0	1 870	28	QJ330-N2-MPA
160	240	38	231 000	335 000	12 300	4750	2 600	6,35	QJ1032-N2-MPA
	290	48	370 000	530 000	20 500	4050	2 200	15,3	QJ232-N2-MPA
170	260	42	280 000	430 000	15 200	4350	2 340	8,79	QJ1034-N2-MPA
	310	52	420 000	630 000	23 500	3750	2 010	18,6	QJ234-N2-MPA
180	280	46	340 000	510 000	19 200	4050	2 1 4 0	11,4	QJ1036-N2-MPA
	320	52	435 000	680 000	24 600	3 600	1 870	19,6	QJ236-N2-MPA
190	290	46	345 000	540 000	19800	3 900	2 010	11,4	QJ1038-N2-MPA
200	310	51	390 000	620 000	22 000	3 600	1 890	15	QJ1040-N2-MPA

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